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SURFACE DURABILITY OF BEVEL GEAR

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This information is valid for bevel gears which are used for power transmission in general industrial machines. The applicable ranges are:

Transverse module / m / 1.5-25mm

Pitch diameter / d0 / 1600mm or less (Straight bevel gear), 1000mm or less (Spiral bevel gear)

Tangential speed / v / 25m/s or less

Rotational speed / n / 3600rpm or less

(1) Basic Conversion Formulas

Equations (10.27), (10.28) and (10.29) in 1.3 "Bending strength of bevel gears" shall apply.

(2) Surface Durability Equations

In order to obtain a proper surface durability, the transmitted tangential force at the central pitch circle, F_{tm} , should not exceed the allowable tangential force at the central pitch circle, F_{tmlim} , based on the allowable Hertz stress σ_{Hlim} .

$$F_{tm} \leq F_{tmlim} \quad (10.40)$$

Alternately, the Hertz stress, σ_H , which is derived from the transmitted tangential force at the central pitch circle should not exceed the allowable Hertz stress, σ_{Hlim} .



BEVEL GEARBOX

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$$\sigma_H \leq \sigma_{Hlim} \quad (10.41)$$

The allowable tangential force at the central pitch circle, F_{tlim} (kgf), can be calculated from Equation (10.42).

$$F_{tlim} = \left(\frac{\sigma_{Hlim}}{Z_M} \right)^2 \frac{d_{01}}{\cos \delta_{01}} \frac{R_a - 0.5b}{R_a} b \frac{i^2}{i^2 + 1} \left(\frac{K_{HL} Z_L Z_R Z_V Z_W K_{HX}}{Z_H Z_\epsilon Z_\beta} \right)^2 \frac{1}{K_{H\beta} K_V K_O} \frac{1}{C_R^2} \quad (10.42)$$

The Hertz stress, σ_H (kgf/mm²), is calculated from Equation (10.43).

$$\sigma_H = \sqrt{\frac{\cos \delta_{01} F_{tm}}{d_{01} b} \frac{i^2 + 1}{i^2} \frac{R_a}{R_a - 0.5b} \frac{Z_H Z_M Z_\epsilon Z_\beta}{K_{HL} Z_L Z_R Z_V Z_W K_{HX}} \sqrt{K_{H\beta} K_V K_O} C_R} \quad (10.43)$$

(3) Determination of Factors

(3)-1 Facewidth, b

This term is defined as the facewidth on the pitch cone. For a meshed pair, the narrower gear's b is to be used.

(3)-2 Zone Factor, Z_H

The zone factor, Z_H, is defined as:

$$Z_H = \sqrt{\frac{2 \cos \beta_g}{\sin \alpha_s \cos \alpha_s}} \quad (10.44)$$

where β_m : Mean spiral angle

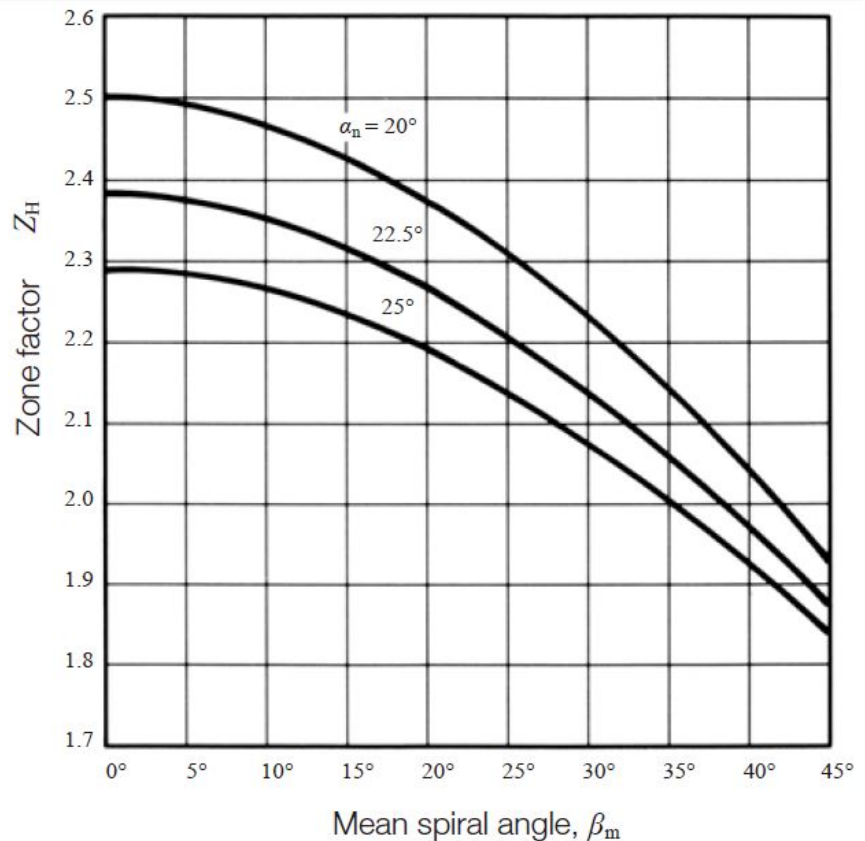
α_n : Normal reference pressure angle

α_s : Central transverse pressure angle = $\tan^{-1} (\tan \alpha_n / \cos \beta_m)$

$\beta_g = \tan^{-1} (\tan \beta_m \cos \alpha_s)$

If the normal reference pressure angle, α_n , is 20 degree, 22.5 degree or 25 degree, the zone factor, Z_H, can be obtained from Figure 10.10.



Fig. 10.10 Zone factor, Z_H (3)-3 Material Factor, Z_M

The material factor, Z_M , can be obtained from Table 10.9 in 1.2 "Surface durability of spur gear and helical gear".

(3)-4 Contact Ratio Factor, Z

The contact ratio factor, Z_ϵ , is calculated from the equations below.

Straight bevel gear : $Z_\epsilon = 1.0$

Spiral bevel gear :

$$\left. \begin{array}{l} \text{When } \epsilon_\beta \leq 1 \quad Z_\epsilon = \sqrt{1 - \epsilon_\beta + \frac{\epsilon_\beta}{\epsilon_\alpha}} \\ \text{When } \epsilon_\beta > 1 \quad Z_\epsilon = \sqrt{\frac{1}{\epsilon_\alpha}} \end{array} \right\} \quad (10.45)$$

Where

ϵ_α : Transverse contact ratio

ϵ_β : Overlap ratio

$$\epsilon_\beta = \frac{R_a}{R_a - 0.5b} \frac{b \tan \beta_m}{\pi m} \quad (10.45a)$$

(3)-5 Spiral Angle Factor, Z_β

Since it is difficult to prescribe the spiral angle factor, Z_β , because little is known about this factor, 1.0 is usually used.

$$Z_\beta = 1.0 \quad (10.46)$$

^

(3)-6 Life Factor, KHL

The life factor for surface durability, KHL, is obtainable from Table 10.10 in 1.2 "Surface durability of spur and helical gear".

(3)-7 Lubricant Factor, ZL

The lubricant factor, ZL, is found in Figure 10.3. See page 673.

(3)-8 Surface Roughness Factor, ZR

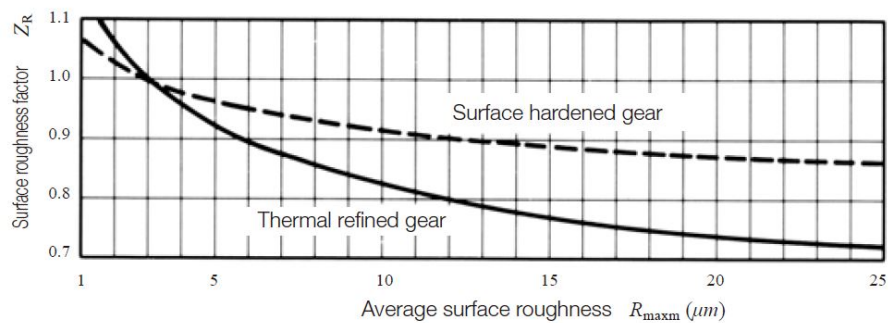
The surface roughness factor, ZR, is obtainable from Figure 10.11 on the basis of average roughness, R_{maxm} (μm). The average surface roughness, R_{maxm} , is calculated by Equation (10.47) from surface roughness of the pinion and gear (R_{max1} and R_{max2}), and a (mm).

$$R_{maxm} = \frac{R_{max1} + R_{max2}}{2} \sqrt[3]{\frac{100}{a}} \quad (\mu m) \quad (10.47)$$

where $a = R_m(\sin \delta_{01} + \cos \delta_{01})$

$$R_m = R_a - b/2$$

Fig. 10.11 Surface roughness factor, ZR

**(3)-9 Lubrication speed factor, ZV**

The lubrication speed factor, ZV, is obtained from Figure 10.5. See page 673.

(3)-10 Hardness ratio factor, ZW

The hardness ratio factor, ZW, applies only to the gear that is in mesh with a pinion which is quenched and ground, and can be obtained from Equation (10.48).

$$Z_W = 1.2 - \frac{H_{B2} - 130}{1700} \quad (10.48)$$

Where H_{B2} : Brinell hardness of the tooth flank of the gear should be $130 \leq H_{B2} \leq 470$

If the gear's hardness is outside of this range, ZW is assumed to be unity.

$$Z_W = 1.0 \quad (10.49)$$



(3)-11 Size Factor, K_{HX}

The size factor, K_{HX} , is assumed to be unity because, often, little is known about this factor.

$$K_{HX} = 1.0 \text{ (10.50)}$$

(3)-12 Longitudinal Load Distribution Factor, $K_{H\beta}$

The longitudinal load distribution factors are listed in Tables 10.25 and 10.26. If the gear and pinion are unhardened the factors are to be reduced to 90% of the values in the table.

Table 10.25 Longitudinal load distribution factor for spiral bevel gears (zerol bevel gears included), and straight bevel gears with crowning, $K_{H\beta}$

Table 10.26 Longitudinal load distribution factor for straight bevel gear without crowning, $K_{H\beta}$



(3)-13 Dynamic Load Factor, KV

The dynamic load factor, KV, can be obtained from Table 10.24. See page 683.

(3)-14 Overload Factor, KO

The overload factor, KO, can be computed by Equation (10.12) or found in Table 10.4.

(3)-15 Reliability Factor, CR

The general practice is to assume CR to be at least 1.15.

(3)-16 Allowable Hertz Stress, σ_{Hlim}

The values of allowable Hertz stress, σ_{Hlim} , are given in Tables 10.12 through 10.16.

(4) Example of Calculation



Gleason straight bevel gear design details



Surface durability factors of Gleason straight bevel gear

Related links:

[Strength and Durability of Gears](#) – A page of The ABC's of Gears / Basic Guide – B

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