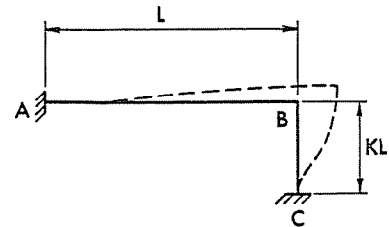
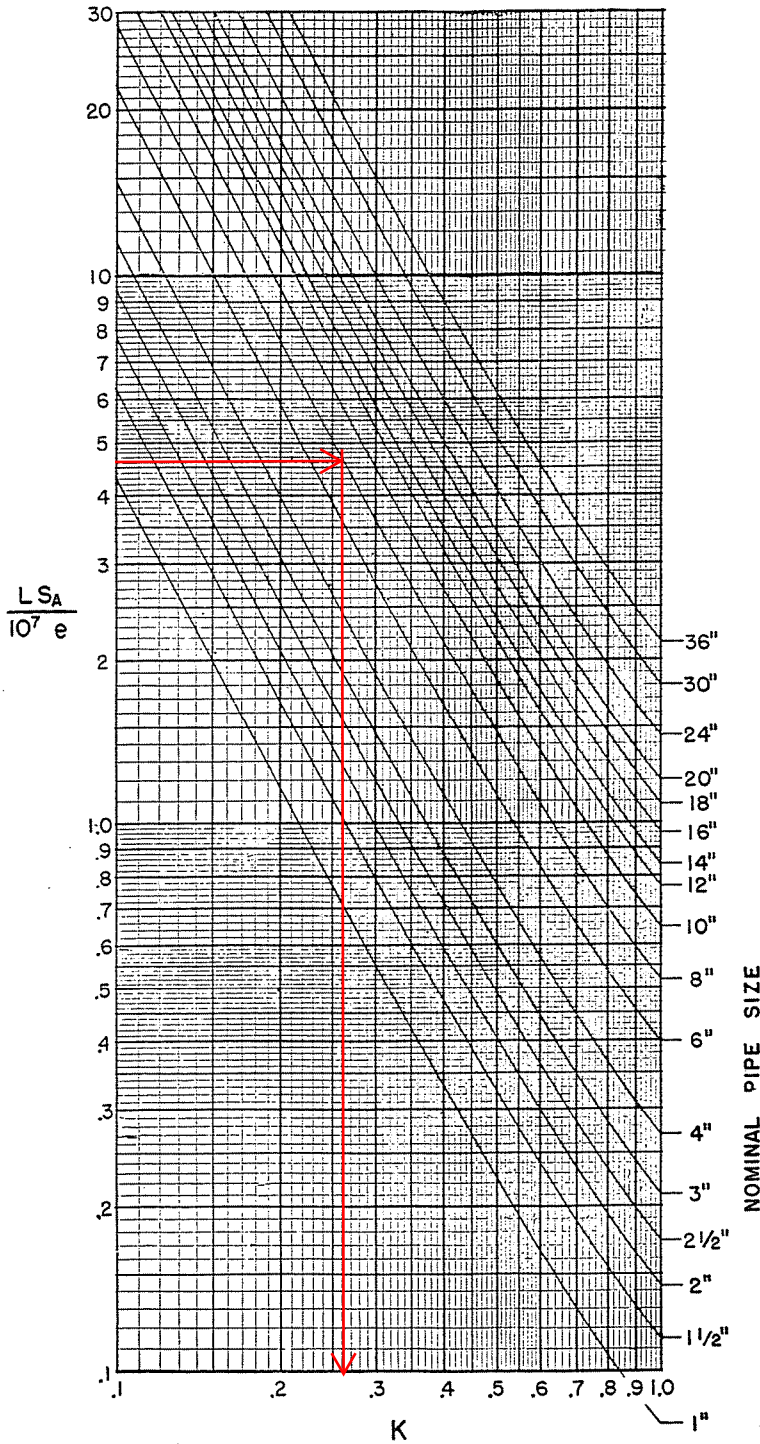


C-5. LENGTH OF LEG REQUIRED

Two-Member System, Both Ends Fixed, Thermal Expansion in Plane of Members



$L$  = Length of leg  $AB$ , ft.

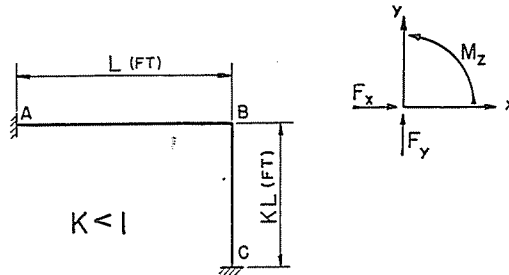
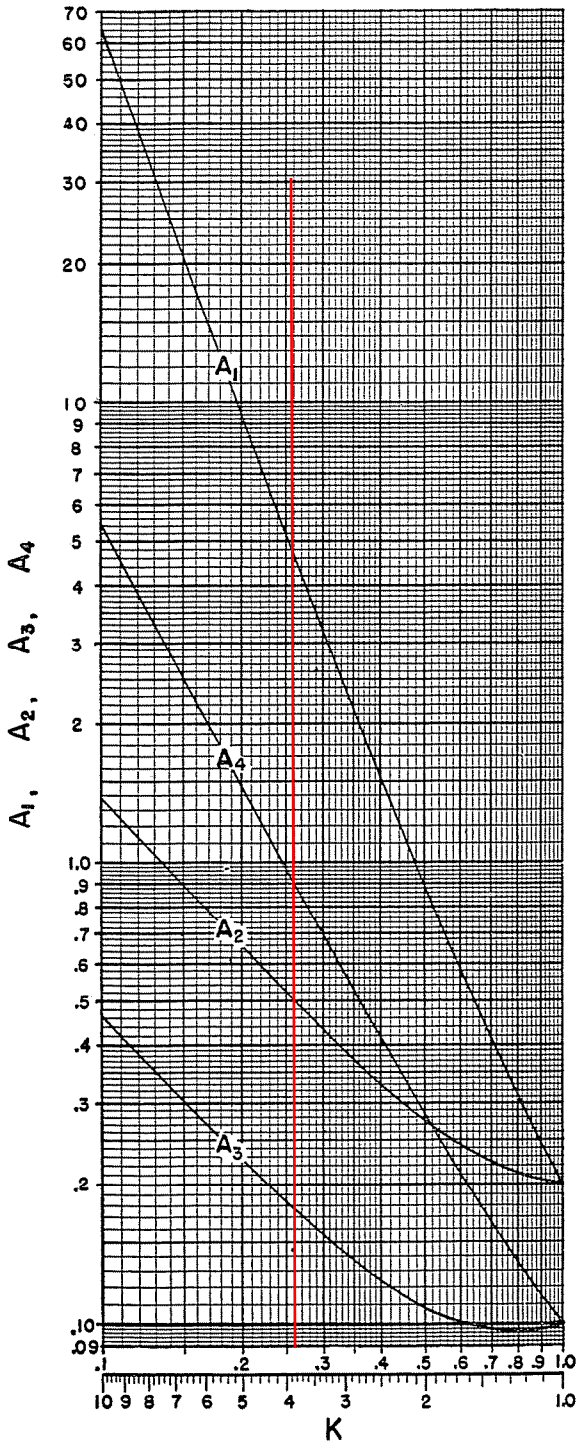
$e$  = Unit linear thermal expansion, in./ft.

Value of  $E$  used =  $29 \times 10^6$  psi.

$S_A$  = Code allowable stress range  
( $1.25S_c + 0.25S_h$ ), psi.

Multiply  $L$  by  $K$  to obtain length of leg  $BC$  required.

**C-6. MOMENTS AND FORCES**  
**Two-Member System, Both Ends Fixed**  
**Thermal Expansion in Plane of Members**

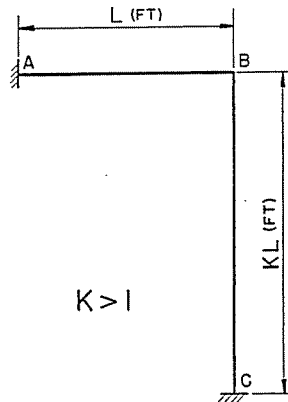


$$F_{xA} = -F_{xC} = -10^6 A_1 I e / L^2$$

$$F_{yA} = -F_{yC} = +10^6 A_2 I e / L^2$$

$$M_{zA} = +10^6 A_3 I e / L$$

$$M_{zC} = -10^6 A_4 I e / L$$



$$F_{xA} = -F_{xC} = -10^6 A_2 I e / (KL)^2$$

$$F_{yA} = -F_{yC} = +10^6 A_1 I e / (KL)^2$$

$$M_{zA} = 10^6 A_4 I e / KL$$

$$M_{zC} = -10^6 A_3 I e / KL$$

$F$  = Force, lb.  
 $M$  = Moment, ft-lb.

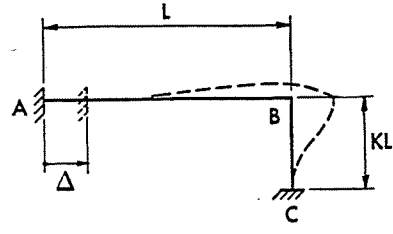
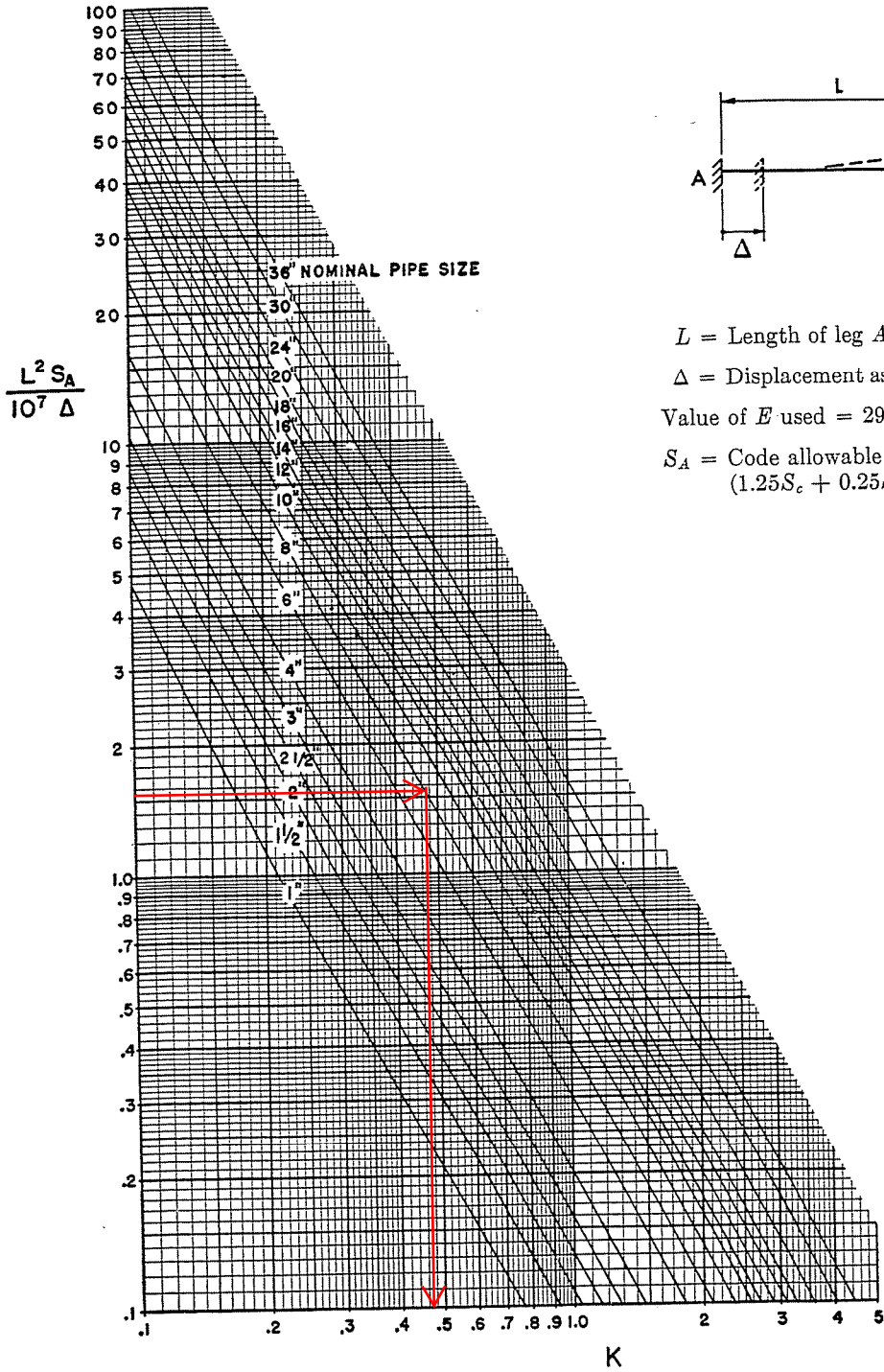
First subscript denotes *direction*.  
 Second subscript denotes *location*.  
 Signs are those of forces or moments *acting on* anchors.

$I$  = Moment of inertia of pipe, in.<sup>4</sup>.  
 $e$  = Unit linear thermal expansion, in./ft.  
 Value of  $E$  used =  $29 \times 10^6$  psi.

DESIGN OF PIPING SYSTEMS

C-7. LENGTH OF LEG REQUIRED

Two-Member System, Both Ends Fixed  
 One Support Displaced in the Direction of the Adjoining Member

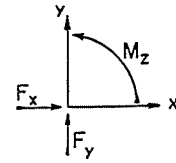
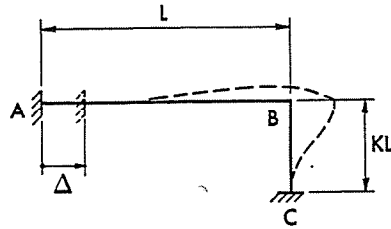
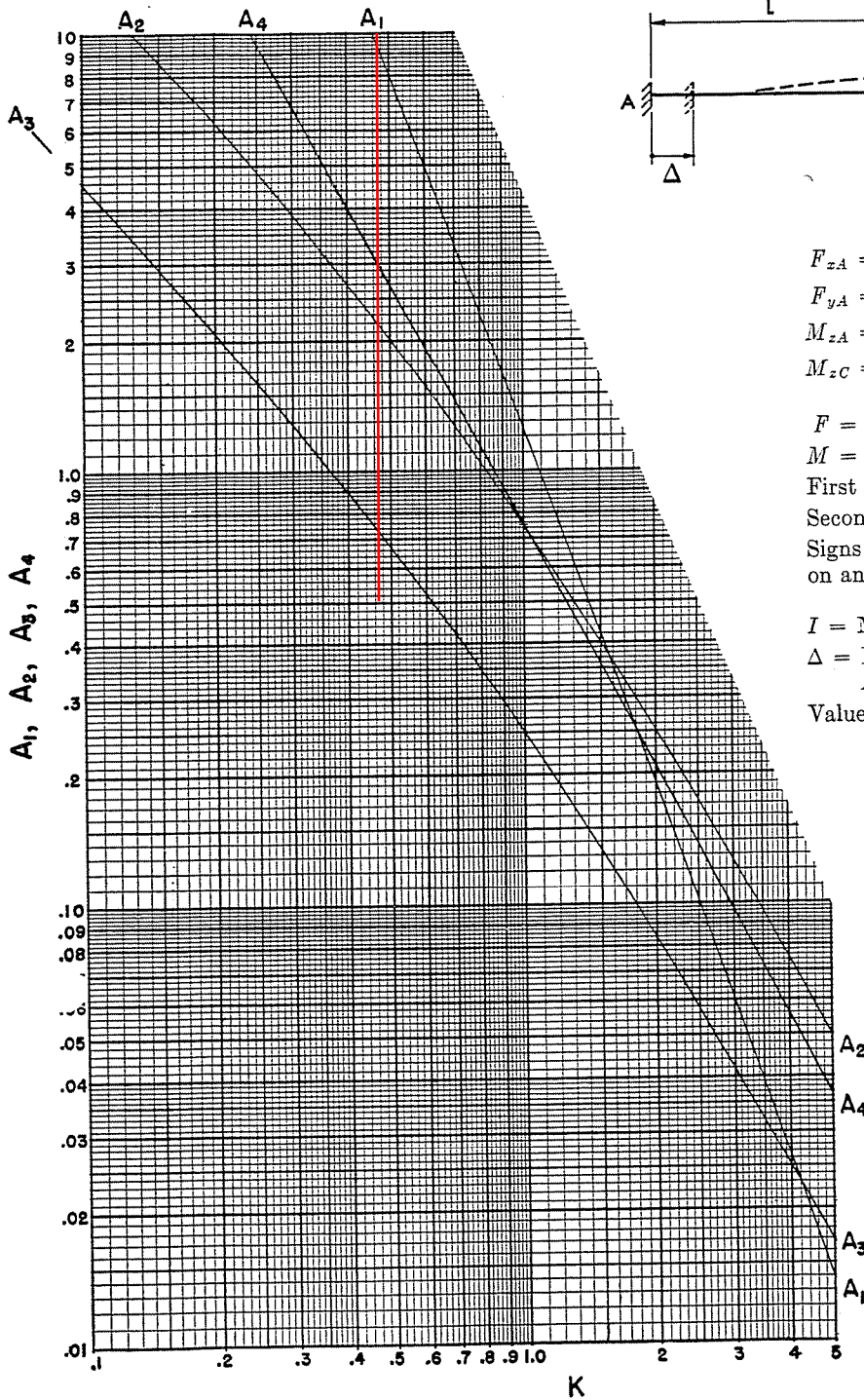


$L$  = Length of leg AB, ft.  
 $\Delta$  = Displacement as shown of end A, in.  
 Value of  $E$  used =  $29 \times 10^6$  psi.  
 $S_A$  = Code allowable stress range  
 ( $1.25S_c + 0.25S_h$ ), psi.

Multiply  $L$  by  $K$  to obtain length of BC required.

C-8. MOMENTS AND FORCES

Two-Member System, Both Ends Fixed  
 One Support Displaced in the Direction of the Adjoining Member



$$F_{zA} = -F_{zC} = -10^5 I \Delta A_1 / L^3$$

$$F_{yA} = -F_{yC} = +10^5 I \Delta A_2 / L^3$$

$$M_{zA} = +10^5 I \Delta A_3 / L^2$$

$$M_{zC} = -10^5 I \Delta A_4 / L^2$$

$F$  = Force, lb.

$M$  = Moment, ft-lb.

First subscript denotes *direction*.

Second subscript denotes *location*.

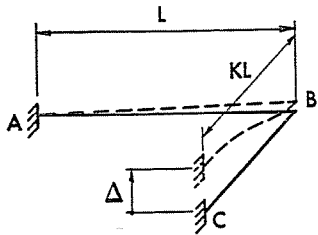
Signs are those of forces or moments *acting* on anchors.

$I$  = Moment of inertia of pipe, in.<sup>4</sup>.

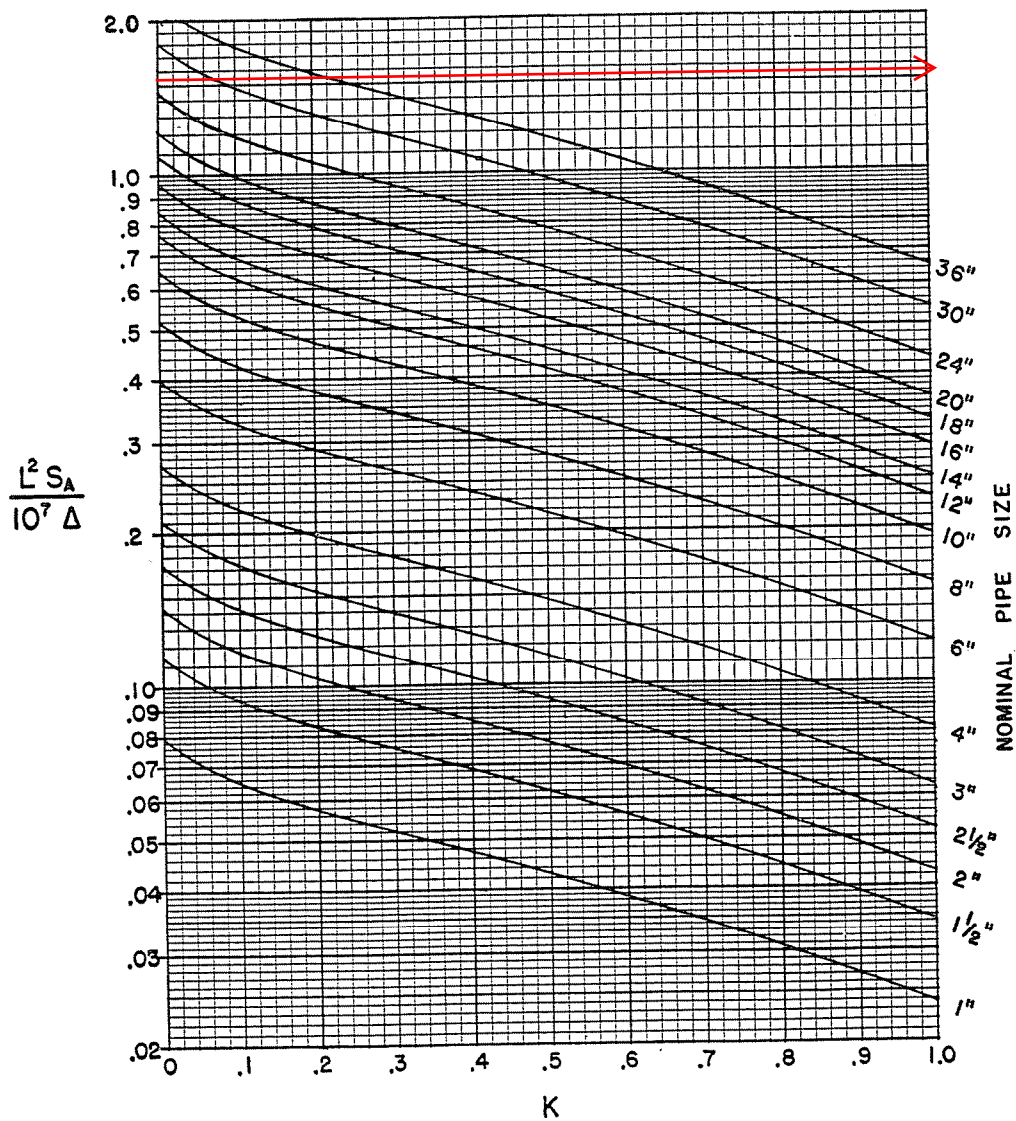
$\Delta$  = Displacement in  $x$ -direction of end  $A$ , in.

Value of  $E$  used =  $29 \times 10^6$  psi.

**C-9. LENGTH OF LEG REQUIRED**  
**Two-Member System, Both Ends Fixed**  
**One Support Displaced Normal to Plane of Members**

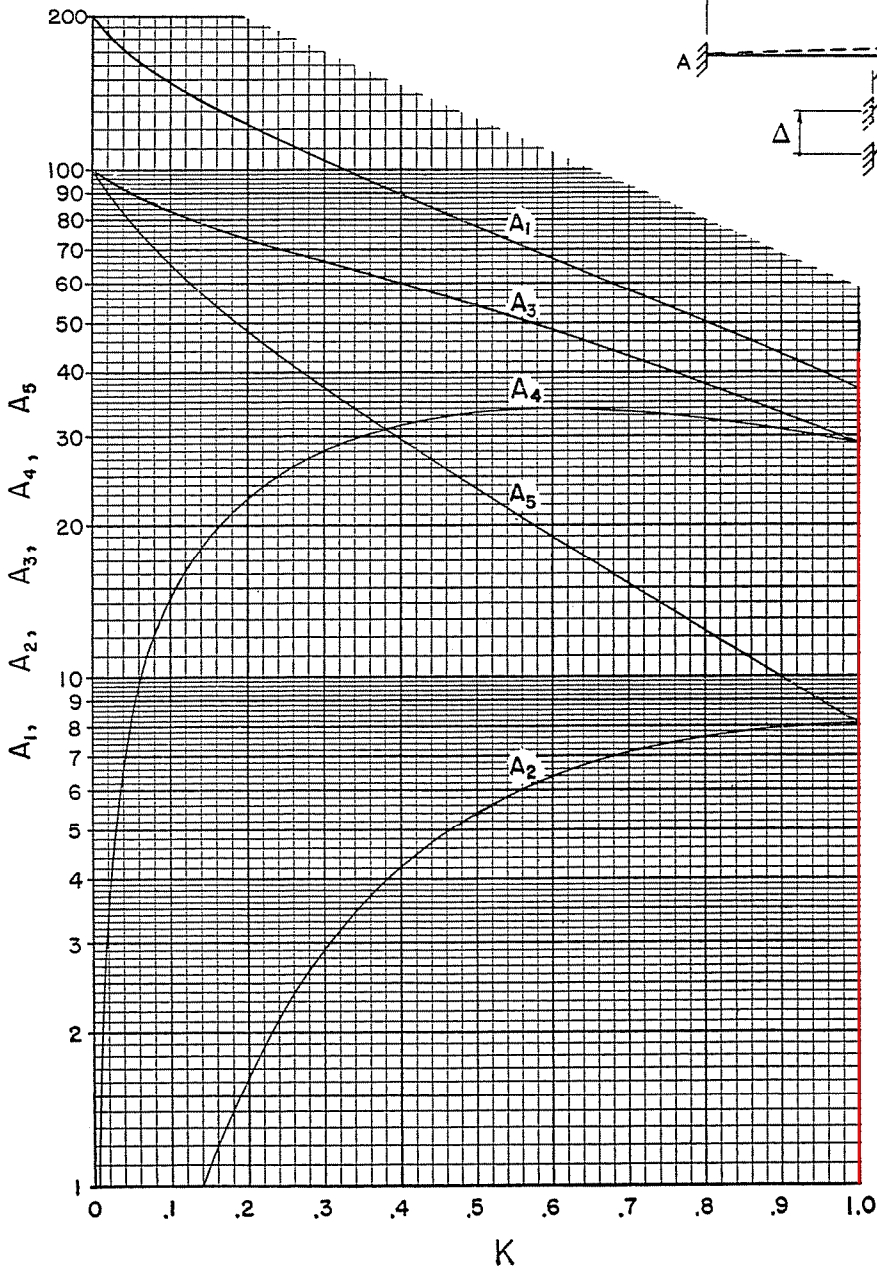
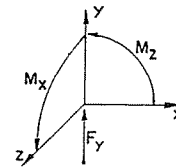
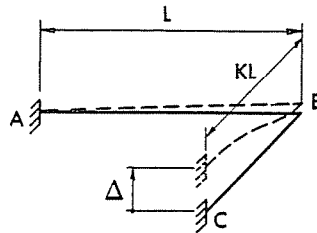


$L$  = Length of leg  $AB$ , ft.  
 $\Delta$  = Displacement normal to plane, in.  
 Value of  $E$  used =  $29 \times 10^6$  psi.  
 $S_A$  = Code allowable stress range ( $1.25S_c + 0.25S_h$ ), psi.



Multiply  $L$  by  $K$  to obtain length of leg  $BC$  required.

**C-10. MOMENTS AND FORCES**  
**Two-Member System, Both Ends Fixed**  
**One Support Displaced Normal to Plane of Members**



$$F_{yA} = -F_{yC} = 10^3 \frac{I\Delta}{L^3} A_1$$

$$M_{xA} = -10^3 \frac{I\Delta}{L^2} A_2$$

$$M_{zA} = +10^3 \frac{I\Delta}{L^2} A_3$$

$$M_{xC} = -10^3 \frac{I\Delta}{L^2} A_4$$

$$M_{zC} = 10^3 \frac{I\Delta}{L^2} A_5$$

$F$  = Force, lb.

$M$  = Moment, ft-lb.

First subscript denotes *direction*.

Second subscript denotes *location*.

Signs are those of forces or moments *acting* on anchors.

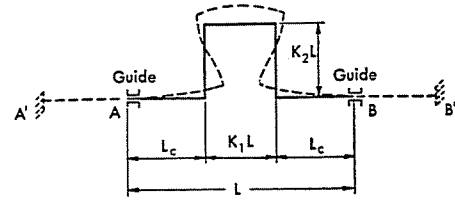
$I$  = Moment of inertia of pipe, in.<sup>4</sup>.

$\Delta$  = Displacement normal to plane, in.

Value of  $E$  used =  $29 \times 10^6$  psi.

DESIGN OF PIPING SYSTEMS

C-11. REQUIRED HEIGHT  
Symmetrical Expansion Loop



$L$  = Length from  $A$  to  $B$ , ft.

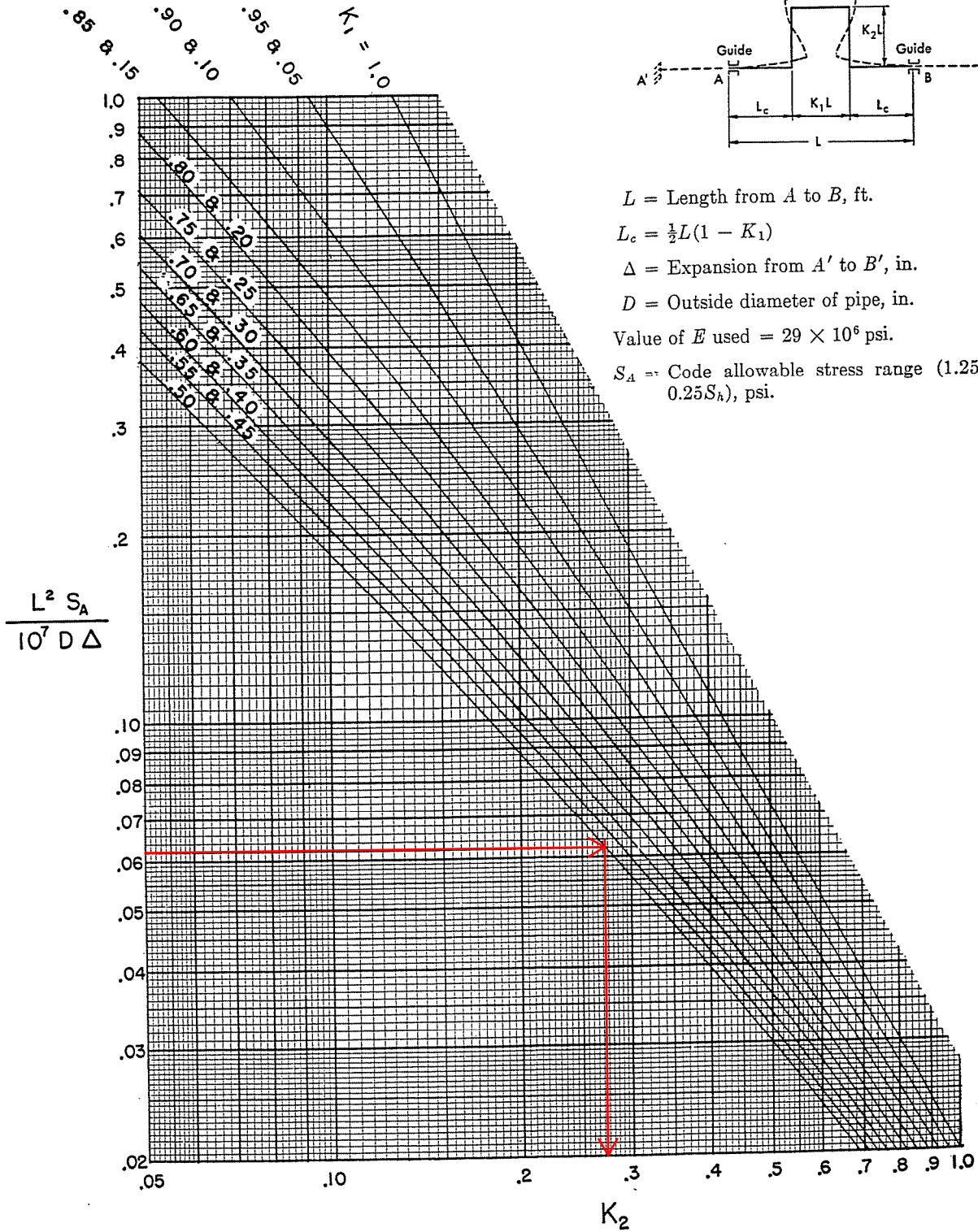
$L_c = \frac{1}{2}L(1 - K_1)$

$\Delta$  = Expansion from  $A'$  to  $B'$ , in.

$D$  = Outside diameter of pipe, in.

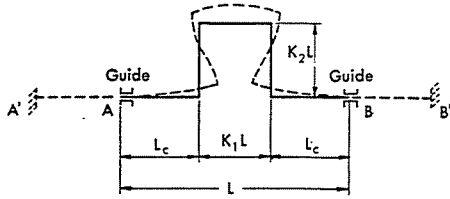
Value of  $E$  used =  $29 \times 10^6$  psi.

$S_A$  = Code allowable stress range ( $1.25S_c + 0.25S_h$ ), psi.



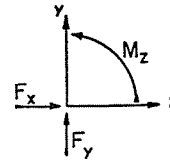
Multiply  $L$  by  $K_1$  and  $K_2$  to obtain dimensions of loop.

**C-12. MOMENTS AND FORCES**  
**Symmetrical Expansion Loop**



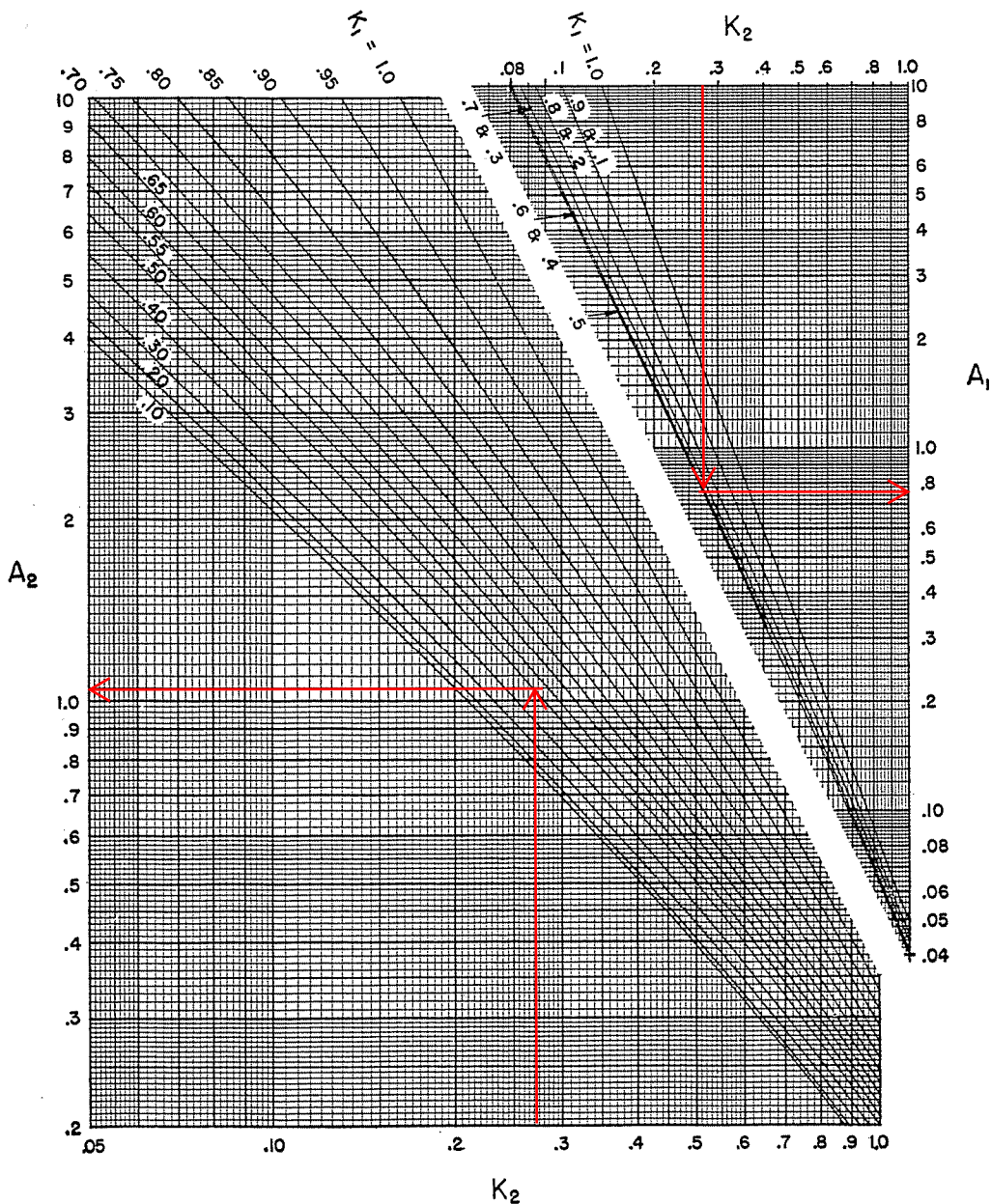
$$F_{xA'} = -F_{xB'} = -10^6 A_1 I \Delta / L^3$$

$$M_{zA} = -M_{zB} = 10^5 A_2 I \Delta / L^3$$



$I$  = Moment of inertia of pipe, in.<sup>4</sup>.  
 $\Delta$  = Expansion from  $A'$  to  $B'$ , in.  
 Value of  $E$  used =  $29 \times 10^6$  psi.

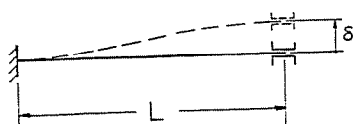
$F$  = Force, lb.  
 $M$  = Moment, ft-lb.  
 First subscript denotes *direction*.  
 Second subscript denotes *location*.  
 Signs are those of forces or moments *acting on anchors*.





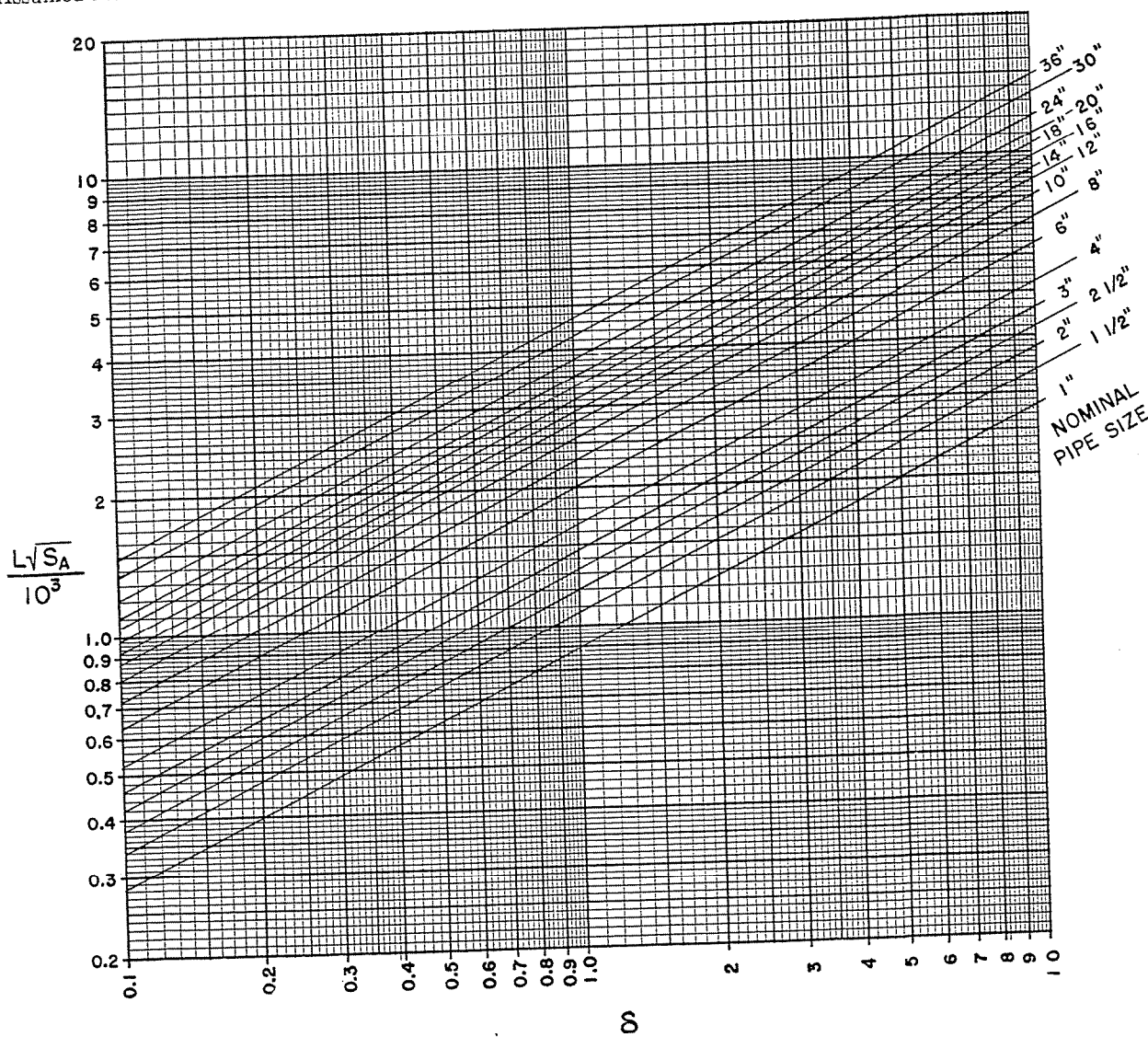
DESIGN OF PIPING SYSTEMS

C-13. Guided Cantilever Chart



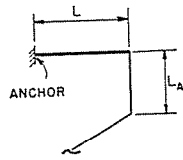
Assumed mode of deflection of guided cantilever.

$L$  = Length of leg, ft.  
 $\delta$  = Lateral deflection, in.  
 Value of  $E$  used =  $29 \times 10^6$  psi.  
 $S_A$  = Code allowable stress range ( $1.25S_c + 0.25S_h$ ), psi.

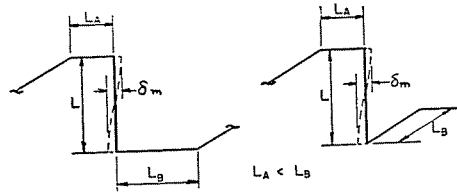


*Instructions:* Determine value of  $L\sqrt{S_A}/10^3$ . Enter with this value of ordinate scale and read over to line for proper nominal pipe size. Read down to abscissa scale. The value obtained will be the permissible lateral deflection for leg.

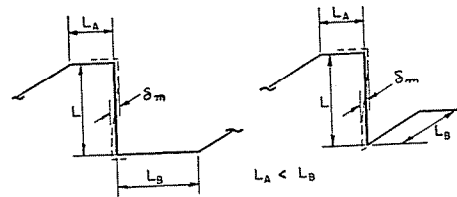
C-14. Correction Factor  $f$ , Guided Cantilever Method



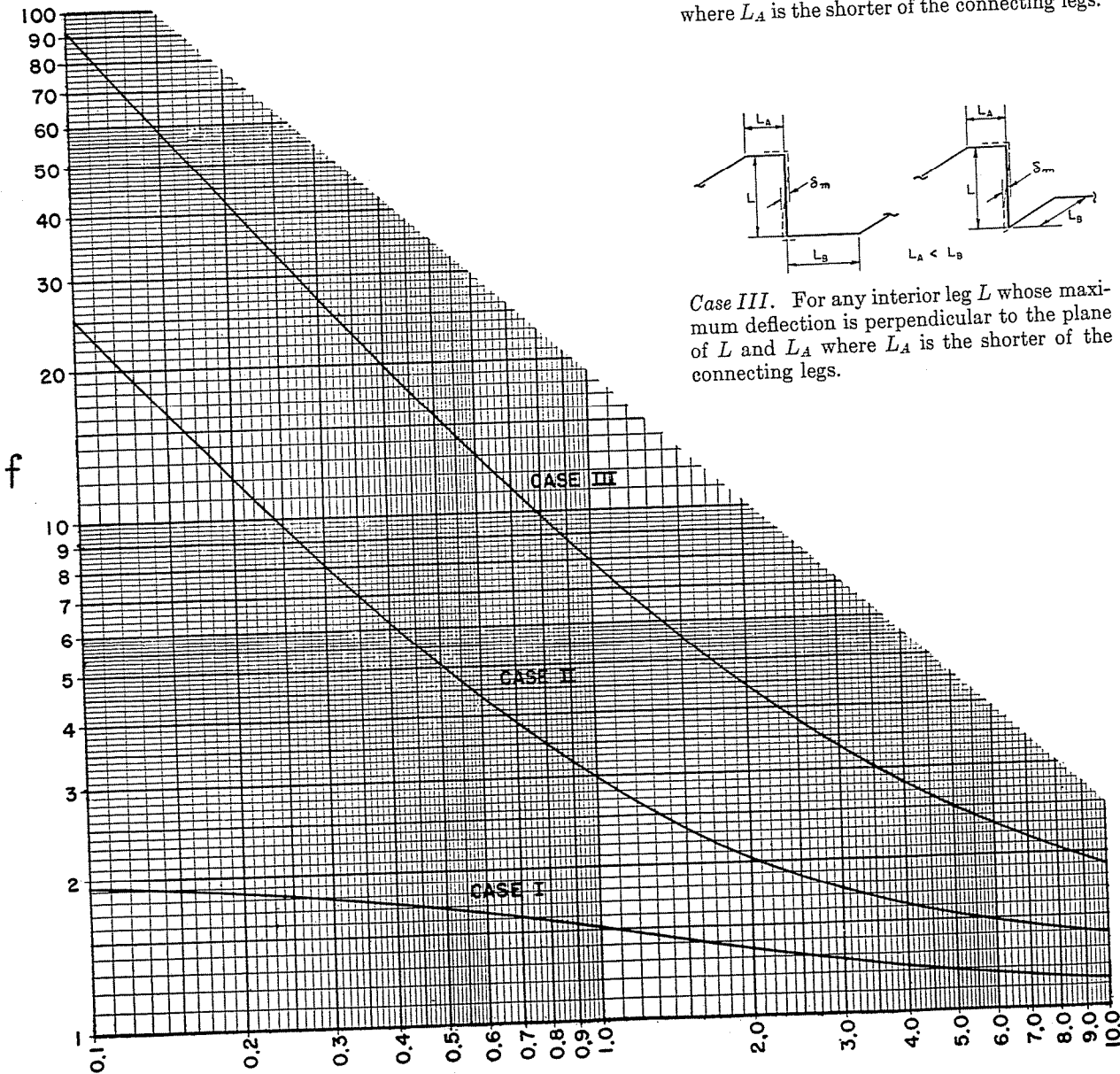
Case I. For any exterior leg  $L$ .



Case II. For any interior leg  $L$  whose maximum deflection is in the plane of  $L$  and  $L_A$  where  $L_A$  is the shorter of the connecting legs.



Case III. For any interior leg  $L$  whose maximum deflection is perpendicular to the plane of  $L$  and  $L_A$  where  $L_A$  is the shorter of the connecting legs.



$$\frac{L}{L_A} = \frac{\text{LENGTH OF LEG UNDER CONSIDERATION}}{\text{LENGTH OF SHORTER ADJACENT LEG}}$$